

A New and Simple Model for the Prediction of Horizontal Well Productivity in Gas Condensate Reservoirs

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ABSTRACT

Horizontal wells are a proven and acknowledged technology to enhance well productivity through an increase in reservoir contact compared to that of vertical wells under the same conditions. In the last three decades, a considerable effort has been directed by many investigators to study flow around horizontal wells. In gas condensate reservoirs, in addition to the three dimensional (3D) nature of the flow geometry, the flow behaviour is further complicated by the phase change and the variation of relative permeability (k_r) due to the coupling (increase in k_r by an increase in velocity or decrease in IFT) and inertia (a decrease in k_r by an increase in velocity) effects. Therefore, simulating such a complex 3D flow using commercial numerical simulators requires a 3D fine grid compositional approach which is very impractical, cumbersome and sometimes triggers convergence problems due to numerical instability. So far, none of these studies propose a method to deal with the complex multiphase behaviour of gas condensate flow around the horizontal well. Consequently, the introduction of a quick and reliable tool for long term productivity calculation in such a system is much needed.

This paper presents a technique which was developed through a comprehensive study of the flow behaviour around horizontal wells in gas condensate reservoirs involving the creation of many in-house mathematical models using finite element and finite difference methods. An in-house simulator was developed to accurately model the multiphase flow of gas and condensate around horizontal wells. A large data bank was then generated covering the impact of a wide range of pertinent geometric and flow parameters on well performance including: well and reservoir geometries, reservoir and bottom-hole pressure, fluid velocity, gas oil ratio and fluid composition.

Based on the results of the simulation, a new method has been proposed to predict the productivity of horizontal wells for the case of multiphase flow of gas and condensate. In this approach, the flow behaviour of gas and condensate around the well is quantified in terms of the effective wellbore radius of an equivalent open hole that replicates flow around the actual 3D system. The effective wellbore radius varies with fluid properties, velocity and interfacial tension (IFT), reservoir and wellbore conditions. The integrity of the new methodology has also been verified for various fluids and flow conditions.

With this approach, a simple spreadsheet, without recourse to complex numerical simulation, can predict the horizontal well performance, significantly facilitating engineering and management decisions on the application of costly horizontal well technologies.

Keywords: gas condensate flow; horizontal wells; effective wellbore radius; well productivity; interfacial tension

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33 1. INTRODUCTION

34 In gas condensate reservoirs, as the pressure falls below the dew point pressure, a bank of condensate forms
35 around the wellbore which affects the flow behaviour and consequently well productivity. Under such condition,
36 two hydrocarbon phases (gas and condensate) co-exist under making the phase and flow behaviour completely
37 different from those of dry gas reservoirs. It is also well documented that the fluid flow behaviour around the
38 wellbore region, of near critical gas condensate systems, characterized by very low interfacial tension, is different
39 from that of conventional oil gas systems. Thus, the relative permeability of gas condensate systems has a unique
40 dependency on interfacial tension (Bardon and Longeron 1980, Asar and Handy 1988) and velocity (Danesh et.
41 al, 1994, Henderson et. al 1996, Ali et. al 1997, Bloom et. al 1997). Accordingly, any reservoir simulator or model
42 proposed for well calculations in such systems must take into account these effects in order to make a sufficiently
43 accurate prediction of the well performance.

44 The reduction of relative permeability permeability at high velocities due to negative inertia (non-Darcy flow)
45 was first introduced by Forchheimer (1914). The dependency of the relative permeability of low IFT systems on
46 interfacial tension was first reported by Bardon (1980). The improvement of the relative permeability of gas and
47 condensate fluids due to an increase in velocity was first reported experimentally by the gas condensate recovery
48 research team of Heriot-Watt University (Danesh et al. 1994, Henderson et al. 1996). The positive coupling effect,
49 which refers to the improvement of relative permeability as velocity increases and/or IFT decreases, has been
50 shown theoretically and experimentally to be due to the simultaneous coupled flow of the gas and condensate
51 phases with intermittent opening and closure of the gas passages by the condensate at the pore level
52 (Jamiolahmady et al. 2000, 2003).

53 The breakthrough of drilling technology in developing and completing horizontal wells has significantly
54 impacted oil and gas reservoir development strategies. Such a well trajectory increases the well exposure to the
55 reservoir drainage area and thereby appreciably increases the well productivity. However, the cost of horizontal
56 wells is often a major barrier. Over the last few years, since horizontal wells have been drilled more economically,
57 the number of horizontal wells has grown exponentially worldwide. Therefore, many studies have been conducted
58 to propose an accurate formulation for the calculation of the productivity of such wells at steady state (SS) or
59 pseudo steady state (PSS) conditions ((Borisov, 1964), (Joshi, 1991), (Giger et al., 1984), (Permadi, 1995),
60 (Renard and Dupuy, 1991), (Lu, 2001), (Elgaghah et al., 1996), (Economides et al., 1996), (Furui et al., 2003),
61 (Wang and Eaton, 2007) , (Sheng and J, 2007), (Escobar and Montealegre, 2008) , (Economides et al., 1996),
62 (Helmy and Wattenbarger, 1998), (Lu and Tiab, 2007), (Babu and Odeh, 1989) and (Goode and Kuchuk, 1991)).
63 However, these equations are only applicable for single phase flow conditions.

64 There are only a few sensitivity studies which have discussed horizontal well performance in gas condensate
65 reservoirs. Numerical simulations conducted by (Hashemi and Gringarten, 2005), (Dehane and Tiab, 2000),
66 (Marir and Djebbar, 2006) and (Jamiolahmady and Danesh, 2007) have all mainly been focused on the sensitivity
67 of pertinent parameters on the performance of such wells.

68 As noted earlier, the flow behaviour of gas condensate reservoirs is very complex due to phase change, inertia
69 and coupling. An accurate estimation of the productivity of such systems using a numerical simulator is a
70 challenging task because this 3D compositional simulation requires a fine grid to capture the abrupt variation of
71 fluid and flow parameters around the wellbore. This is cumbersome and impractical for field applications.

72 This work is devoted to study flow behaviour around a horizontal well by developing a 3D horizontal well
73 two phase compositional (gas condensate) mathematical simulator under steady state conditions, and proposing a
74 model to predict the horizontal well productivity for such systems. In this work, we used a generalized correlation
75 for kr calculation in the in-house simulator developed by Jamiolahmady et al. 2009. A brief description of the
76 formulations accounting for coupling and inertial effects on kr is explained in Appendix A. The main advantages
77 of this correlation compared to other formulations available in the literature are that it uses either universal
78 parameters or those parameters that can be estimated from readily available petrophysical data and it also accounts
79 for the combined effect of inertia and coupling.

80 In our proposed approach, using the results of the developed 3D horizontal well in-house simulator, an
81 effective wellbore radius is defined for an equivalent open hole radial system replicating the two phase flow of
82 gas and condensate around the horizontal well. As demonstrated here, this formulation and methodology
83 appropriately correlate the impact of pertinent parameters including IFT, velocity, phase changes and the well
84 geometries parameters, i.e. wellbore radius, reservoir thickness and horizontal well length, to the results of the
85 exact solution obtained by the corresponding in-house simulator operating under the same flowing conditions.
86 The proposed formulation, which is simple and easy to use, correctly describes single phase non-Darcy (inertial)
87 flow systems when total gas fractional flow is unity and if velocity is low to Darcy flow conditions.

88

89 **2. FLOW SIMULATION**

90 In this study, we embarked on a modelling approach and developed different mathematical simulators using
91 the Comsol multi-physics mathematical package, which is based on finite element methods. The finite element
92 technique has been proved to be a suitable choice for the complex flow geometry, and is widely applied by many
93 authors ((Furui et al., 2003), (Jamiolahmady et al., 2005), (Karakas and Tariq, 1991); (Klotz et al., 1974)). It
94 should be noted that simulating such three-dimensional well geometry using a commercial compositional
95 simulator for the large number of simulations required to cover the required wide range of variation of pertinent
96 parameters proved to be quite cumbersome. We also experienced a lot of convergence problems due to numerical
97 instability when we initially considered a commercial reservoir simulator. The adaptive mesh generator of the
98 Comsol mathematical package was tuned to generate a fine mesh that secures the accuracy and precision of the
99 results.

100

101 ***Three-Dimensional Horizontal Well In-House Simulators***

102 The three-dimensional system considered in this study consists of a horizontal well with wellbore radius (r_w)
103 and length (L), in a single layer cubic reservoir, as shown in Figure 1. This homogenous porous medium has an
104 absolute permeability (k) and a formation thickness (h). The length of the reservoir in the x and y directions is
105 assumed to be 2.5 times the length of the horizontal well. It has been shown (Ghahri P., 2010) that the impact of
106 partial penetration on the geometric skin and the productivity of horizontal wells is negligible for $X_{res}/L > 1.5$.

107 The equations employed in this study are similar to those described for gas condensate flow in a perforated
108 region (Jamiolahmady et al., 2007) as presented in Appendix B. The methodology and boundary conditions are
109 also described in Appendix B.

110 Due to the existing symmetry, only a quarter of the model has been considered in this study. This reduces the
111 computation time and facilitates having a high quality mesh. Mesh quality is the most important parameter of
112 mesh design, which affects both the accuracy and convergence of a numerical solution. Considerable attempts
113 were made to generate consistent high quality mesh files for different 3D horizontal well geometries. We have
114 secured the accuracy of the results of our horizontal well models by selecting a high quality mesh as well as
115 quadratic interpolation shape functions, which approximate the partial differential equation.

117 ***Reservoir Fluid***

118 The composition and fluid properties of equilibrated phases of a fixed overall composition depend only on
119 the pressure for a given temperature. In this study, binary moderate (0.74 of C1 (methane) & 0.26 of n-C4 (normal
120 butane)) and rich (0.88 of C1 and 0.12 of C10 (decane)) gas condensate fluid models were used to validate the
121 integrity of the proposed approach. The maximum liquid drop out of the Constant Volume Depletion (CVD) test
122 of these moderate and rich gas condensate fluid models are 23% and 30%, respectively. The values of
123 composition, density (ρ), viscosity (μ) and interfacial tension (IFT) of the C1-nC4 mixtures which were
124 implemented in the model are those measured in the gas condensate group laboratory as well as literature data
125 ((Sage et al., 1940); SUPERTRAPP (Ely and Huber, 1992); (Weinaug and Katz, 1943)) at 311 K over a wide
126 pressure range. For the selected C1-C10 rich gas condensate fluid model, we used the PVTi option of the
127 ECLIPSE simulator to generate the PVT data.

129 ***Validation Of The Horizontal Well In-House Simulator***

130 The accuracy of the two phase mathematical in-house simulator (Comsol) was confirmed by comparing some
131 of its results with those of ECLIPSE300 (GeoQuest) at the same prevailing flow conditions.

132 The reservoir model in this exercise had the rock properties of Texas Cream with porosity of 0.21 and
133 permeability of 9.1 mD. The reservoir fluid was a binary mixture of C1 (methane) and n-C4 (normal butane) as
134 described in the previous section. The reservoir dimensions were 38 m in the x and y directions and 4 m in the z
135 direction. The horizontal well length was 15 m. The ECLIPSE and Comsol reservoir models are presented in
136 Figure 2. Many different cases were simulated using both the ECLIPSE300 (GeoQuest) and the in-house
137 simulators. The fractional flow at average reservoir pressure was the same in both simulators. A very fine grid
138 was used to capture the abrupt changes in the flow parameters near the wellbore. As the k_r correlations used in
139 these two models (i.e. Comsol & ECLIPSE300) are different, the base curve relative permeability was used to
140 describe flow around the horizontal well. This is the relative permeability curve measured at a high IFT (above
141 which k_r is independent of coupling) and low velocity (below which k_r is independent of coupling and inertial
142 effects).

143 In the ECLIPSE 300 model, seventy injection wells were placed at the boundary of the reservoir to keep the
144 reservoir pressure constant at the drainage boundary.

145 Figure 3 shows the good agreement between the two results. The arithmetic average absolute percentage
146 deviation (AAD%) of the flow rates predicted by the ECLIPSE300 simulator (GeoQuest) compared to those of
147 the horizontal well simulator was 2.9 %.

149 3. A NEW MODEL (AN IDEAL OPEN HOLE) FOR HORIZONTAL WELL 150 PRODUCTIVITY CALCULATION

151 A horizontal well usually experiences less pressure drop compared to a vertical open hole system. This
152 pressure change is usually quantified in terms of a skin, or an effective wellbore radius, included in the well
153 productivity calculation of an equivalent open hole system. The combined effects of all the skin factors due to
154 geometry, formation damage, perforation and high velocity (non-Darcy), etc. is referred to as a total skin, S_t .

155 Therefore a horizontal well productivity is expressed by

$$156 \quad Q = \frac{2\pi kh\Delta p}{\text{Ln}\left(\frac{r_e}{r_w}\right) + S_T} = \frac{2\pi kh\Delta p}{\text{Ln}\left(\frac{r_e}{r_w'}\right)}. \quad (1)$$

157 Here, the drainage volumes of the horizontal and its equivalent open hole system are assumed to be equal ($V_h =$
158 V_v).

159 The effective wellbore radius of an open hole is defined here as follows:

$$160 \quad r_w' = r_w \times e^{-S_T}. \quad (2)$$

161 where S_t is the total skin. For single phase Darcy flow without formation damage, the total skin is equal to
162 the geometric skin, $S_t = S_g$.

163 In an earlier study, the author developed a correlation for geometric skin (S_g), which correctly accounts for
164 the impact of horizontal well geometry on the flow, using the efficient statistical method response surface method
165 (P. Ghahri et al. 2009). The integrity and accurately of the proposed formulation was validated against a developed
166 mathematical numerical model and other formulations available in the literature, which have limited range of
167 application. Accordingly, the formula is very reliable for a wide range of variation of pertinent parameters.

168 However, it should be noted that the above formulation is only applicable for single phase flow horizontal
169 well productivity and is not capable of explicitly predicting the productivity of two phase flow of gas and
170 condensate or single phase non-Darcy flow. The computation of the effective wellbore radius is not
171 straightforward because of the difference in the impact of the phase change and the velocity and IFT effects on
172 the performance of the vertical and horizontal two flow systems. Therefore, the effective wellbore radius for such
173 cases, should not only be a function of wellbore geometry, but also an implicit function of volumetric flow rate
174 and fluid properties.

175 In this study, the main objective is to propose a formulation for the effective wellbore radius, which accounts
176 for the combined effects of geometrical parameters (i.e. wellbore radius, reservoir thickness and horizontal well
177 length) as well as flow parameters (i.e. IFT, velocity and fluid properties) on the horizontal well productivity.

178 The proposed semi analytical model predicts the horizontal well productivity of single phase non-Darcy flow
179 and two phase flow of gas and condensate based on an equivalent open hole model with an effective wellbore
180 radius.

181 The pseudo pressure concept is used to extend the single phase horizontal well productivity calculation to the
182 two phase flow of gas and condensate.

$$m = \frac{2\pi k h \Delta m(p)}{\text{Ln}\left(\frac{r_e}{r_w}\right) + S_T} = \frac{2\pi k h \Delta m(p)}{\text{Ln}\left(\frac{r_e}{r_w}\right)}, \quad (3)$$

where $m(p)$ is the pseudo pressure function defined for two phase flow of gas and condensate as:

$$m(p) = \int_{p_w}^{p_e} \left(\frac{\rho_g k_{rg}}{\mu_g} + \frac{\rho_c k_{rc}}{\mu_c} \right) dp. \quad (4)$$

Here, m is the mass flow rate, p_w and p_e are the wellbore and external pressures. ρ , μ and k_r are the density, viscosity and relative permeability. The subscripts g and c refer to gas and condensate.

As will be explained later, a formulation for the effective wellbore radius has been proposed here which accounts for the combined effects of the well geometry, IFT, velocity and fluid properties on the horizontally well productivity.

The details of solving Equation 3 for an open hole using finite difference method and pseudo pressure has been described in Appendix C. This solution can be easily done using an excel spread sheet for different number of grids.

195 **Effective Wellbore Radius Formulation**

In a systemic approach, a model is first proposed, which simulates the steady state single phase non-Darcy flow, including the inertial effect on the well productivity calculation. Then the proposed model is extended for a two phase of gas and condensate.

200 **Single Phase Non-Darcy Flow**

In the in-house simulator the two phase k_r (k_{rg} and k_{rc}) formulation is converted to single phase non Darcy by substituting $GTR=1$ in the equation A-7, as explained in Appendix A. The two phase pseudo pressure function in Equation 3 is also converted to the single phase pseudo pressure function as follow:

$$m(p) = \int_{p_w}^{p_e} \left(\frac{\rho_g k_{rg}}{\mu_g} \right) dp \quad (4)$$

where k_{rg} for single phase non Darcy is:

$$k_{rg} = \frac{1}{1 + \text{Re}} \quad (5)$$

Re is the Reynolds number defined by:

$$\text{Re} = \frac{\rho |V| k \beta}{\mu} \quad (6)$$

ρ , V , k , β and μ refer to the single-phase density, velocity, permeability, inertia factor and viscosity.

210 It should be noted that applying Equations 6 and 5 in Darcy Law will correctly convert the Darcy equation to
 211 the Forchheimer equation, which describes single phase non-Darcy inertial flow. In other words, the inertial effect
 212 is defined as a single phase relative permeability (Equation 5). As the velocity decreases, for very long horizontal
 213 wells for instance, the single phase relative permeability (k_{rg}) approaches to 1, reducing the equation to the Darcy
 214 flow equation. However, for predicting the productivity of short to medium length horizontal wells, where the
 215 inertia reduces the productivity significantly, k_r would be less than one and the effective well bore radius should
 216 be corrected to account for the high velocity inertial effect (P. Ghahri 2010).

217 To obtain an appropriate formulation for r_w' , the mass flow rates of the 1D equivalent open hole model, EOH,
 218 (Equation 3) and those of 3D horizontal well in-house simulators were matched iteratively by varying r_w' .

219 It is proposed that a combination of geometric skin, the two dimensionless numbers (defined in Equations 7
 220 and 8) and the single phase inertial factor express the impact of all pertinent parameters.

$$221 \quad L_D = \frac{L}{h} \quad (7)$$

$$222 \quad h_D = \frac{h}{r_w} \quad (8)$$

223 $L, h,$ and r_w are the horizontal well length, reservoir thickness and wellbore radius.

224 Further analysis showed that the effective wellbore radius can be represented by Equation 9.

$$225 \quad r_w' = \frac{r_w \times e^{-S_g}}{1 + a \times \left(\frac{Re \times h_D}{L_D} \right)^b} \quad (9)$$

226 S_g is the geometric skin and can easily be calculated assuming single phase gas Darcy flow around the
 227 horizontal well using any available formulation in the literature (i.e. Joshi, Borosiv, Economides, Goode, P. and
 228 Kuchuk, Babu, D. and Odeh, Ghahri et. al (2009), etc). The above equation is very simple and easy to use, i.e.
 229 there are only two coefficients (a and b) that need to be estimated. Here, two sets of data were generated based on
 230 the basic physical properties of two different rocks used to confirm the integrity of the proposed formulation. The
 231 effective wellbore radius varied substantially when the reservoir permeability was varied from 11 to 110 mD, for
 232 the same pressure drop, as the velocity was significantly higher for the latter. The coefficients a and b were
 233 estimated as 0.04 and 0.6 respectively, using one of these sets of data; i.e. the set of data referred to as HW-1 in
 234 Table 1. The other data set, referred to as HW-2 in Table 1, was used to verify the reliability of the correlation as
 235 described below.

236 Figure 4 confirms the accuracy of the developed flow skin equation by comparing the calculated mass flow
 237 rate obtained using an equivalent open hole, in which the developed r_w' formulation, Equation 9, and the iterative
 238 procedure described in Appendix C have been incorporated, with those of the horizontal well in-house simulator.
 239 The reservoir models and the range of velocities are those listed in Table 1, the HW-1 data set. The AAD%
 240 (average absolute deviation error) for 270 data points in this study is only 2%.

241 The accuracy of the developed effective wellbore radius formulation was further verified by applying the
 242 proposed method to another a reservoir model, with different permeability and velocity with AAD% of 2%. Here,
 243 Texas Cream core properties with permeability 11 mD described the reservoir model. A wide range of variation

244 of horizontal well length, wellbore radius and thickness of the reservoir were covered for the 220 data points as
 245 shown in the HW-2 data set (Table 1).

246

247 ***Two Phase Flow Of Gas and Condensate***

248 In this part, we followed an approach similar to that mentioned above and proposed a general formulation for
 249 the calculation of the effective wellbore radius of an equivalent open hole system for two phase flow of gas
 250 condensate, which correctly extends to that of single phase when the gas fractional flow is unity (GTR=1) and
 251 which produces the same flow performance, as that of a 3D horizontal well system.

252 To obtain an appropriate formulation for r_w' , the mass flow rates of the two phase open hole model (Equation
 253 3) and that of 3D two phase horizontal well in-house simulators were matched iteratively by varying r_w' .

254 This was done for a large bank of data, containing over 1300 data points. For this part of study, the Texas
 255 Cream and Berea described the reservoir rock properties. The two fluids were binary mixtures of C1 and C4 and
 256 C1 and C10 introduced above. The variation of the reservoir pressure, wellbore pressure and GTR (Total gas
 257 fractional flow) are given in Table 2. The other characteristic of different models used here are also listed in Table
 258 2.

259 For a gas condensate system, the flow regime is controlled by coupling, operating at moderate to high
 260 velocities, and inertia, operating at high velocities. These two act in opposite directions, i.e. the former improves
 261 the flow performance whilst the latter reduces the flow efficiency.

262 Figures 5 and 6 show pressure and condensate saturation distribution maps in a drainage area of a horizontal
 263 well. It should be noted that the employed generalized k_r correlation (Jamiolahmady et al. 2009) generates k_r at
 264 different velocity and IFT, with the relative permeability ratio ($k_{rgtr}=k_{rg}/(k_{rg}+k_{rc})$) rather than local saturation as
 265 the main independent. However, the values of saturation as a function of k_{rgtr} are known for the base k_r curve.
 266 Considering that saturation values do not enter into the pseudo-pressure calculation, these saturation values are
 267 assigned to the corresponding k_{rgtr} at any time and location: that is, the saturation map across the reservoir is
 268 calculated by knowing k_{rgtr} across the reservoir. It is clear that the shapes of condensate and pressure profile
 269 distributions are similar.

270 For two phase flow of gas and condensate, the important parameters, which affect well productivity, are
 271 density, viscosity and relative permeability of both flowing (gas and condensate) phases. Mass mobility, expressed
 272 by Equation 10, is used to express the effect of these three important parameters on the effective wellbore radius.

$$273 \quad \bar{M}_r = \frac{\rho_g k_{rg}}{\mu_g} + \frac{\rho_c k_{rc}}{\mu_c}, \quad (10)$$

274 where $\frac{\rho_g k_{rg}}{\mu_g}$ and $\frac{\rho_c k_{rc}}{\mu_c}$ are the absolute mass mobility for gas and condensate, respectively.

275 We intend to use an average value of mass mobility to modify and extend Equation 9 to gas condensate
 276 systems. Here we multiply Equation 9 by the ratio of (two phase gas and condensate) mass mobility of the actual
 277 flow to that of the base case. For this latter mass mobility term, the base relative permeability is used. Hence,
 278 Equation 5 can be rewritten as:

$$279 \quad r_w' = \frac{r_w \times e^{-Sm}}{1 + a \times \frac{\text{Re} \times h_D}{L_D}} \times \left(\frac{\bar{M}_{rEOH}}{\bar{M}_{rHW})_{Darcy}} \right)_{ave} = \frac{r_w \times e^{-Sm}}{1 + a \times \frac{\text{Re} \times h_D}{L_D}} \times \left(\frac{\bar{M}_{rEOH}}{\bar{M}_{rEOH})_{Darcy}} \right)_{ave}, \quad (11)$$

280 where \bar{M}_{rEOH} and $\bar{M}_{rEOH})_{Darcy}$ are the average mass mobility for EOH with and without considering the
281 coupling and inertial effects, respectively. Or in another form, Equation 11 can be rewritten as:

$$282 \quad r_w' = \frac{r_w \times e^{-Sm}}{1 + a \times \left(\frac{\text{Re} \times h_D}{L_D} \right)^b} \times \left(\frac{\left(\frac{\rho_g k_{rg} + \rho_c k_{rc}}{\mu_g \mu_c} \right)_{EOH}}{\left(\frac{\rho_g k_{rg} + \rho_c k_{rc}}{\mu_g \mu_c} \right)_{EOH-Darcy}} \right)_{ave} = \frac{r_w \times e^{-Sm}}{1 + a \times \left(\frac{\text{Re} \times h_D}{L_D} \right)^b} \\ \times \left(\frac{\left(\frac{\rho_g k_{rg} + \rho_c k_{rc}}{\mu_g \mu_c} \right)_{EOH}}{\left(\frac{\rho_g k_{rgb} + \rho_c k_{rcb}}{\mu_g \mu_c} \right)_{EOH}} \right)_{ave} \quad (12)$$

283 The average mobility ratio term depends on the fluid properties and relative permeability across the reservoir
284 drainage area. It is important to point out that the above formulation includes the gas and condensate relative
285 permeability and hence the impact of velocity and IFT in the form of coupling and inertial effects has been
286 included. We have noted that this formula expresses the horizontal well performance accurately, but it is possible
287 to make it simpler for practical purposes.

288 Based on a pseudo-pressure calculation approach, the condensate relative permeability can be estimated by
289 the definition of fractional flow using the following equation (Jamiolahmady et. al. 2006):

$$290 \quad k_{rc} = k_{rg} \times \frac{1 - GTR}{GTR} \times \frac{\mu_c}{\mu_g}, \quad (13)$$

291 where GTR, μ_c and μ_g are the total gas fraction, condensate and gas viscosity, respectively. Substituting the
292 above equation into Equation 11 results in:

$$293 \quad r_w' = \frac{r_w \times e^{-Sm}}{1 + a \times \left(\frac{\text{Re} \times h_D}{L_D} \right)^b} \times \left(\frac{\left(\frac{k_{rg}}{\mu_g} \left(\rho_g + \rho_c \times \frac{1 - GTR}{GTR} \right) \right)_{EOH}}{\left(\frac{k_{rgb}}{\mu_g} \left(\rho_g + \rho_c \times \frac{1 - GTR}{GTR} \right) \right)_{EOH}} \right)_{ave}, \quad (14)$$

294 Rearranging Equation 14 gives:

$$295 \quad r_w' = \frac{r_w \times e^{-Sm}}{1 + a \times \left(\frac{\text{Re} \times h_D}{L_D} \right)^b} \times \left(\frac{\left(\frac{k_{rg}}{\mu_g \times GTR} \left(\rho_g \times GTR + \rho_c \times (1 - GTR) \right) \right)_{EOH}}{\left(\frac{k_{rgb}}{\mu_g \times GTR} \left(\rho_g \times GTR + \rho_c \times (1 - GTR) \right) \right)_{EOH}} \right)_{ave}, \quad (15)$$

296 or

$$r_w' = \frac{r_w \times e^{-Sm}}{1 + a \times \left(\frac{\text{Re} \times h_D}{L_D} \right)^b} \times \left(\frac{\left(\frac{k_{rg}}{\mu_g \times GTR} (\bar{\rho}_{ave})_{EOH} \right)_{ave}}{\left(\frac{k_{rgb}}{\mu_g \times GTR} (\bar{\rho}_{ave})_{EOH} \right)_{ave}} \right), \quad (16)$$

where $\bar{\rho}_{ave}$ is the volumetric average density of two phase gas and condensate which is defined by:

$$\bar{\rho}_{ave} = \rho_g \times GTR + \rho_c \times (1 - GTR), \quad (17)$$

The fluid properties and GTR variation in the reservoir depend on the pressure profile for a fluid with fixed total composition. We assume that the pressures and GTR values are the same at the wellbore and exterior boundaries for both the base case and that affected by coupling and inertia. Furthermore, it has previously been shown that the effect of fluid properties on the performance of a perforated well compared to that of an equivalent open hole well, both flowing under same pressure drop and with the same GTR, is minimal (Jamiolahmady et al. 2005). This simplifying assumption implies that the mobility ratio can approximately be estimated by the following equation

$$\left(\frac{\left(\frac{k_{rg}}{\mu_g \times GTR} (\bar{\rho}_{ave})_{EOH} \right)_{ave}}{\left(\frac{k_{rgb}}{\mu_g \times GTR} (\bar{\rho}_{ave})_{EOH} \right)_{ave}} \right) \cong \frac{(k_{rg})_{EOH_{ave}}}{(k_{rgb})_{EOH_{ave}}}, \quad (18)$$

Figures 5 and 6, show the base and affected (by coupling and inertia) gas relative permeability for an equivalent open hole systems for horizontal well-3 and horizontal well-4, defined in Table 2. It is noted that gas relative permeability (k_{rg}) values change mostly around the wellbore, where coupling and inertia are important. Far away from the wellbore, the difference between these two curves, k_{rgb} and k_{rg} , is almost constant. Thus

$\frac{(k_{rg})_{EOH_{ave}}}{(k_{rgb})_{EOH_{ave}}}$ could be approximately predicted by an average value corresponding to that around the wellbore.

Therefore the following equation can be used to estimate the effective wellbore radius:

$$r_w' = \frac{r_w \times e^{-Sm}}{1 + a \times \left(\frac{\text{Re} \times h_D}{L_D} \right)^b} \times \frac{k_{rg}}{k_{rgb}}. \quad (19)$$

It should be noted that Equation 19 can be applied for both single phase and two phase, gas and condensate, flow cases. That is, for the single phase flow case, $k_r=1.0$ and Equation 19 converts to Equation 9. As noted earlier, the above proposed formulations for the effective wellbore radius (Equations 19) and pseudo pressure function (Equations 3 and 4) are both functions of fluid properties and flow rates. Consequently, the productivity calculation requires an iterative procedure. The details of this procedure have been explained in Appendix C.

Figures 7 and 8 confirm the accuracy of the proposed approach by comparing the calculated mass flow rates obtained using an equivalent open hole radius based on the proposed approach with those of the 3D horizontal

322 well in-house simulator. In this exercise wide ranges of the GTR, pressure and horizontal well geometric variables
323 values were considered, as listed in Table 2.
324

325 **7. APPLICATION OF NEW MODEL FOR THE HORIZONTAL WELL** 326 **PRODUCTIVITY PRODUCTION**

327 The added value of the proposed approach has been demonstrated by studying the impact of pertinent
328 parameters; well length, reservoir thickness and gas fractional flow on horizontal well productivity using an excel
329 spreadsheet (1D simulator) which implements the new model formulation (as explained in Appendix D). For all
330 cases presented here, the fluid was C1-nC4 and the reservoir rock properties were those of HW-1 are cases as
331 listed in Table 2. The reservoir drainage and wellbore radius are 200 and 0.1 m, respectively. The differential
332 pressure across the reservoir model is 300 psi, unless otherwise stated.

333 It should be noted that, performing the simulation corresponding to each point shown in Figures 9 to 11, using
334 3-D compositional simulator as an alternative tool will be quite time consuming and cumbersome, especially if
335 there are convergence problems.
336

337 ***Well Length***

338 For these simulations, the pressure at the wellbore and the differential pressure across the reservoir model
339 were 1500 and 300 psi, respectively. For all the cases presented here, the well productivity is calculated both with
340 and without coupling and inertia (referred to as NI and NCI, respectively). It should be noted that, for the well
341 productivity without coupling and inertia, the base curve relative permeability is used. As can be seen from Figure
342 9, the well productivity increases when the effect of inertia and coupling are taken into consideration. Furthermore,
343 coupling has a more pronounced effect on well productivity for a shorter horizontal well.
344

345 ***Reservoir Thickness***

346 The effect of reservoir thickness on the productivity of a horizontal well with CI effects is shown in Figure
347 10. The reservoir, well dimensions, rock and fluid properties used here are those for Figure 9.

348 It is noted that the well productivity is greater for the thicker reservoir model; that is, the drainage volume for
349 flow of a thick reservoir model compared to that in a thin reservoir is much greater. Furthermore, as the horizontal
350 length (L) increases, the difference between the mass flow rates for thick and thin reservoirs increases
351 substantially.
352

353 ***Gas Fractional Flow***

354 Figure 11 shows the effect of gas to total flow rate ratio (GTR_w) on the productivity of a horizontal well, with
355 CI effects. For the set of the productivity calculations presented in Figure 11, the horizontal well length is 50 m.
356 As can be seen, increasing GTR_w decreases the well productivity, due to the more pronounced effect of inertia.

357 **7. SUMMARY AND CONCLUSIONS**

358 A number of in-house simulators were developed to simulate the performance of horizontal wells in gas and
359 gas condensate reservoirs for steady state conditions. Using the results of these simulators, a generalized effective
360 wellbore radius formulation for an equivalent open hole system was developed which replicate the flow
361 performance of the corresponding complete 3-D system accounts for coupling, inertia and compositional effects.
362 The accuracy of this approach was confirmed by comparing the results with corresponding values obtained using
363 the 3-D horizontal well in-house simulator.

364 Due to the presence of velocity in this formulation, its implementation requires an iterative procedure.
365 However, the small number of iterations required confirms the validity of the assumptions and the reliability of
366 the proposed approach.

367 The ease and added value of the application of the proposed approach has been demonstrated by studying the
368 impact of well length, reservoir thickness and gas fractional flow using an excel spreadsheet that performs
369 implement this calculation. The results, which are similar to those of the 3D simulator, albeit with much less
370 calculation, demonstrate that: (i) coupling has a more pronounced effect on well productivity for the horizontal
371 well with shorter length, (ii) well productivity is greater for the thicker reservoir, especially for the longer
372 horizontal well, and (iii) increasing gas fractional flow decreases the well productivity, due to the more
373 pronounced effect of inertia.

374 These results confirm that the proposed method can be used to estimate horizontal well productivity and
375 optimize well characteristics for the two phase flow of gas and condensate using a simple excel spreadsheet, which
376 makes it attractive for practical purposes.

377

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- 452

453 Appendix A- Generalised Relative Permeability Expression

454 In this section, a summary of the k_r correlation used in the current work is presented. More detailed information
 455 on the structure of the correlation and derivation of formulations can be found elsewhere (Jamiolahmady et al,
 456 2009).

457 The gas relative permeability, k_{rg} , is interpolated between the base curve and the miscible-fluids curve using
 458 an interpolation function Y_g :

$$459 \quad k_{rg} = Y_g (k_{rgb})_{iner} + (1 - Y_g) (k_{rgm})_{iner}. \quad (A-1)$$

460 The miscible and the base curve in Eq. A-1 are based on the relative permeability ratio as the main variable. The
 461 relative permeability ratio is defined as

$$462 \quad k_{rgtr} = \frac{k_{rg}}{k_{rg} + k_{rc}} = \frac{1}{1 + \left(\frac{k_{rc}}{k_{rg}} \right)}. \quad (A-2)$$

463 Solving Eq. A-2 for condensate relative permeability (k_{rc}) gives

$$464 \quad k_{rc} = \frac{k_{rg} (1 - k_{rgtr})}{k_{rgtr}}. \quad (A-3)$$

465 Therefore, when k_{rg} is determined as a function of k_{rgtr} , this equation automatically gives the
 466 corresponding k_{rc} at the same k_{rgtr} .

467 It is noted that the k_{rgtr} ratio could be calculated in terms of gas fractional flow (GTR), using the extended
 468 form of the Darcy's law for two phase flow, without the need for further information.

$$469 \quad k_{rgtr} = \frac{k_{rg}}{k_{rc} + k_{rg}} = \frac{\mu_g GTR}{\mu_g GTR + \mu_c (1 - GTR)}, \quad (A-4)$$

470 The gas fractional flow, GTR, is the ratio of the gas to the total (gas plus condensate) volumetric flow rate, Eq.
 471 A-5.

$$472 \quad GTR = \frac{Q_g}{Q_g + Q_c} = \frac{V_g}{V_c + V_g} \quad (A-5)$$

473 GTR can be expressed in terms of CGFR using Eq. A-6.

$$474 \quad GTR = \frac{1}{1 + CGFR} \quad (A-6)$$

475 The base curve relative permeability curve is adjusted for the effect of inertia using the following equation:

$$476 \quad (k_{rgb})_{iner} = \left(\frac{(k_{rgb})_{meas}}{1 + \frac{\beta [\rho_{avg}]_b k (k_{rgb})_{meas} |V|_T}{GTR \mu_{gb}}} \right) \quad (A-7)$$

477 where ρ_{ave} is the average density based on the fractional flow of the two flowing phases. This equation has
 478 been obtained based on the momentum transfer rate for an equivalent phase, i.e. inertial pressure drop was

479 calculated using the single phase inertial factor, total velocity and total (gas plus condensate) momentum inflow
 480 (Jamiolahmady et al. 2006 and 2009).

481 In Eq. A-7, $(k_{rgb})_{meas}$ is the measured gas relative-permeability at specified base IFT of ≥ 3 mNm⁻¹ (below which
 482 k_r is a function of IFT) and capillary number of $\leq 1E-7$ (above which k_r is a function of velocity). Hereafter when
 483 we refer to base conditions we mean $\sigma_b = 3$ and $N_{cb} = 1E-7$. It is understood that if $IFT > 3$ and $N_{cb} < 1E-7$ the base
 484 relative permeability would be the same. The base velocity in the laboratory test for measuring k_{rgb} should be
 485 selected such that:

$$486 \quad N_{cb} = \frac{\mu_{gb}(V_{gb} + V_{cb})}{\sigma_b} \quad (A-8)$$

487 The miscible gas relative permeability curve, which is modified to include the inertial effect, is calculated as
 488 follows:

$$489 \quad (k_{rgm})_{iner} = \left(\frac{k_{rgtr}}{1 + \beta \rho_m \left(\frac{k}{\mu_m} \right) \|V\|_T} \right) \quad (A-9)$$

490 Similarly to Eq. A-7, this equation has been obtained based on the momentum transfer rate. If Eq. A-9 is added
 491 to the corresponding one for the condensate, one would obtain a k_r term, which if included in Darcy Law gives
 492 the Forchheimer equation, Eq. 2, for the single phase flow conditions.

493 In Eq. A-9, the required miscible fluid properties (i.e. density, ρ_m , and viscosity, μ_m) are the arithmetic average
 494 between the fluid properties of gas and liquid at any given pressure, in the vicinity of the miscible pressure.

495 It should be noted that, because of the presence of the rock properties (k and β) in Eqs. A-7 and A-9, there are
 496 different base and miscible relative permeability vs. k_{rgtr} curves for each core at any velocity value.

497 **Fig. A-1** shows the conventional base (corresponding to $IFT = 3$ mNm⁻¹ and $N_{cb} = 1E-7$, Eq. A-8) and miscible k_{rg}
 498 (45° degree diagonal line) and the corresponding $(k_{rgb})_{iner}$ and $(k_{rgm})_{iner}$ values affected by inertia, which have been
 499 calculated by Eqs. A-7 and A-9, respectively, at a velocity of 82.6 md⁻¹ for the RC3 core. It should be noted that
 500 because of the presence of the rock properties (k and β) in Eqs. A-7 and A-9, different $(k_{rgb})_{iner}$ and $(k_{rgm})_{iner}$ vs.
 501 k_{rgtr} curves at any velocity value, will be obtained for a different porous medium.

502 It should also be noted that although Eqs. A-7 and A-9 are for $S_{wi} = 0$, the same relations are valid for $S_{wi} > 0$, with
 503 $k_{eg}(S_{wi})$ and $\beta_g(S_{wi})$ replacing k and β , respectively.

504 The interpolation parameter, (Y_g) , mainly depends on the rock properties (i.e. k , ϕ , β), interfacial tension (IFT),
 505 pressure gradient, GTR, and the base capillary number as described in the original reference (Jamiolahmady et al.
 506 2009).

507 The interpolation parameter (Y_g) is expressed by Eq. A-10.

$$508 \quad Y_g = \frac{1 + A_1 x}{1 + A_1 x + A_2 x^2}, \quad (A-10)$$

509 with

$$x = \log[\sigma_r Ncr'] \quad A_1 = -\frac{C_1 + C_3 A_3^{C_4}}{C_2 + A_3^{C_4}} \quad A_2 = \frac{C_1 + C_5 A_3^{C_6}}{C_2 + A_3^{C_6}} \quad A_3 = \beta \sqrt{k}$$

$$C_1 = 8, \quad C_2 = 800, \quad C_3 = 0.20, \quad C_4 = 2.20, \quad C_5 = 0.15, \quad C_6 = 0.81$$

where σ_r refers to the ratio of base IFT (σ) to current IFT value and Ncr' refers to the ratio of Nc' , Eq. 5, at any condition to the corresponding value for the base curve at the same k_{rgr} . The base value of Nc' at any k_{rgr} value can be obtained by Eq. A-11, which requires the measured k_{rgb} , fluid properties and the corresponding base Nc of $1E-7$ defined earlier (Eq. A-8),

$$Ncb' = \frac{GTR}{\phi(k_{rgb})_{meas}} Ncb = \frac{GTR}{\phi(k_{rgb})_{meas}} 1E-7, \quad (A-11)$$

It is noted that Y_g mainly depends on the rock properties (i.e. k , ϕ , β), interfacial tension (IFT), pressure gradient, GTR, and the base capillary number. It should be mentioned that in the Y_g expression there is another term, which accounts for the effect of micro-pores. The mercury porosimetry Pc curve of this core indicated that the contribution of micropores was minimal. Therefore, the corresponding additional term to express micro-pore effect has not been included here in Eq. A-10.

It is noted that Jamiolahmady et. al. (2006) had reported a k_r correlation for gas condensate systems, which was based on fractional flow rate and expressed the combined effect of coupling and inertia. However, the dependency of relative permeability to fluid viscosity and lack of a proper definition of the lower limit of the correlation limited its use to the range of IFT values studied. That is, the reported base IFT of 0.85 mNm^{-1} (above which k_r is not a function of IFT), was not a universal value and the threshold capillary number fixing the lower limit of velocity (below which k_r is not a function of velocity) was not identified. Furthermore, the inclusion of viscosity in the definition of independent variable made it difficult to fix the base conditions. In other words, the base k_r curve reported as a function of fractional flow rate would be fluid dependent, i.e. it would vary with the pressure at which the measurements are conducted. This would contradict the definition of the base k_r curve, which should be independent of pressure and velocity at which the measurement is conducted, as described above.

Appendix B- Governing Flow Equations

The equations employed in this study are similar to those described for gas condensate flow in a perforated region (Jamiolahmady et al., 2007).

For a single phase compressible fluid under steady state conditions the governing equation is:

$$\nabla \cdot \left(2 \left[\frac{k}{\mu} \right] \frac{\rho \nabla p}{A} \right) = 0 \quad (B-1)$$

$$A = 1 + \sqrt{1 + 4\rho\beta \left(\frac{k}{\mu} \right)^2 |\nabla p|}$$

where p , μ , k , ρ and β are pressure, viscosity, absolute permeability, density and single phase inertial factor.

539 As noted by (Jamilahmady et al., 2006) the term A is obtained by relating the absolute velocity term $|v|$ to
 540 the pressure gradient by solving the second order polynomial (Forchheimer, 1914a) equation for positive $|v|$ in the
 541 case of a single phase linear Darcy flow system, A is equal to one.

542 The equations describing the two phase flow of gas and condensate around a horizontal well are:

$$543 \quad \nabla \cdot \left(\left[\frac{\rho k_r}{\mu} \right]_g + \left[\frac{\rho k_r}{\mu} \right]_c \right) k \nabla P = 0. \quad (\text{B-2})$$

544 The total fluid composition (z_j) is constant as the fluid flows through the porous media. However, for each
 545 component, there is mass transfer between two phases as expressed by the following equation:

$$546 \quad z_j = \frac{\rho_g y_j GTR + \rho_c x_j (1 - GTR)}{\rho_g GTR + \rho_c (1 - GTR)} = \text{cons.}, \quad (\text{B-3})$$

547 where GTR is the total gas fraction ratio defined by Equation A-4.

$$548 \quad GTR = \frac{Q_g}{Q_g + Q_c} = \frac{1}{1 + \frac{k_{rg}}{k_{rc}} \times \frac{\mu_c}{\mu_g}}. \quad (\text{B-4})$$

549 where Q is the volumetric flow rate and g and c refer to gas and condensate.

550 In Equation A-2, relative permeability which varies with interfacial tension (i.e. pressure for a given fluid
 551 composition) and velocity is estimated using the correlation by (Jamilahmady et al., 2009). In this correlation,
 552 gas relative permeability is interpolated between a base curve and a miscible curve, both corrected for the effect
 553 of inertia, using a generalised interpolation function. The correlation has either universal parameters or those
 554 parameters that can be estimated from readily available petrophysical data. The correlation is based on the relative
 555 permeability ratio ($k_{rgr} = k_{rg}/k_{rg} + k_{rc}$) as the main variable, which is closely related to fractional flow. The
 556 condensate relative permeability is calculated using the definition of relative permeability ratio. It should be noted
 557 that in gas/condensate systems, fractional flow is directly related to fluid composition and pressure at steady-state,
 558 which is generally prevailing near the wellbore, hence making it much more attractive practically, compared to
 559 saturation, which depends on core characteristics.

560

561 **Mathematical Solution Technique**

562 The governing non-linear partial differential equation (PDE), Equations A-1 and A-2 for single phase and
 563 two phase flow, is solved using Comsol multi-physic software (version 3.4, 2007), which uses the finite element
 564 method. The main dependent variable in this equation is pressure (p).

565 The boundary conditions applied to this system are:

- 566 1) Infinite conductivity for the wellbore (the pressure gradient in the wellbore has been ignored),
- 567 2) The pressure at outer boundary (external radius) is known,
- 568 3) The pressure at the inner boundary (wellbore radius) is known.

569 For the two phase flow of gas condensate the equations are solved for both p and GTR. The boundary
 570 conditions applied to the two phase system are those used for the single phase mathematical model as well as

571 implying the constant total composition across the two phase region, so either the GTR or the total fluid
572 composition is known.

573

574 **Appendix C-Numerical Solution for Two Phase Flow**

575 *Numerical Solution for an Equivalent Open Hole Model for Two Phase Flow of Gas*

576 *Condensate:*

577 *Governing Equations*

578 For the radial model, a combination of continuity and non-Darcy flow equations (Equation A-
579 4) can be written as follows:

$$580 \nabla \cdot \left[\frac{1}{r} \left[\frac{k}{\mu} \right] \left[\left(\frac{\rho k_r}{\mu} \right)_g + \left(\frac{\rho k_r}{\mu} \right)_c \right] \cdot \frac{\partial p}{\partial r} \right] = 0. \quad (\text{C-1})$$

581 The differential equation of above equation (B-1) for steady state pressure distribution is:

582

$$\begin{aligned} & \frac{r_{i+1/2}}{\Delta r^2} \times \left(\left(\left(\frac{\rho k_r}{\mu} \right)_g + \left(\frac{\rho k_r}{\mu} \right)_c \right) \times \frac{k}{\mu} \right)_{i+1} \times P_{i+1} - \left(\frac{r_{i+1/2} + r_{i-1/2}}{\Delta r^2} \right) \times \left(\left(\left(\frac{\rho k_r}{\mu} \right)_g + \left(\frac{\rho k_r}{\mu} \right)_c \right) \times \frac{k}{\mu} \right)_i \times P_i + \\ & \frac{r_{i-1/2}}{\Delta r^2} \times \left(\left(\left(\frac{\rho k_r}{\mu} \right)_g + \left(\frac{\rho k_r}{\mu} \right)_c \right) \times \frac{k}{\mu} \right)_{i-1} \times P_{i-1} = 0 \end{aligned}$$

583 . (B-2)

$$584 z_j = \frac{\rho_g y_j GTR + \rho_c x_j (1 - GTR)}{\rho_g GTR + \rho_c (1 - GTR)} = cons \quad (\text{C-2})$$

585

586 For steady state conditions:

$$587 P_1 = P_w$$

$$588 P_n = P_{res} \quad (\text{C-3})$$

589 where subscripts w and res refer to wellbore and external radius.

590

591 **Appendix D-Iterative Procedure for Effective Wellbore Radius Estimation**

592 As noted in the text, the effective wellbore radius formulations depend on the velocity, IFT and fluid
593 properties. Thus, the calculation of the effective wellbore radius needs an iterative procedure, as described below:

594 1) First the effective wellbore radius is estimated based on the geometric skin:

$$595 r_w' = r_w \times e^{-Sg}. \quad (\text{D-1})$$

596 It should be noted that S_g is the geometric skin for the Darcy flow regime and can be easily calculated
597 using any available horizontal well productivity formulation i.e. Joshi, Borosiv, Economides, Goode, P.
598 and Kuchuk, Babu, D. and Odeh, etc.

599 2) Based on the effective well bore radius, the pressure profile and flow rate are calculated using Equation
600 3. It should be noticed that the pseudo pressure calculation is a function of relative permeability or in
601 other words of velocity; therefore an iterative procedure is required to estimate the pressure profile and
602 the mass flow rate.

603 3) Using Equation 11 or 18, the effective wellbore radius is calculated.

604 4) If the difference between the new effective open hole wellbore radius calculated and the previous one is
605 not negligible, the calculation is repeated from step 2, otherwise the calculated effective well bore radius
606 and mass flow rate are reported.

607